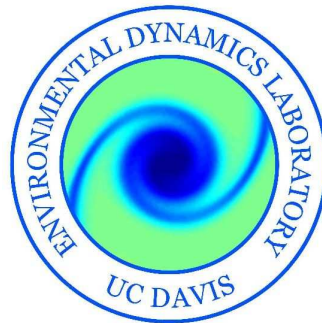


# **An Embedded Boundary Adaptive Mesh Refinement Method for Environmental Flows**

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Stanford University  
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## Talk Outline

- **Introduction**
- **Governing equations**
- **Discretization concepts**
- **Grid generation**
- **Results**
- **Conclusions**

## Introduction

What types of environmental flows do we hope to analyze with this method?

- Highly non-linear, multi-scale flows in oceans, lakes, and rivers
- Flows that are well approximated by the variable density incompressible Navier-Stokes equations
- Examples: internal waves, coastal plumes, density currents in lakes, flows in branched estuarine slough networks, flows past highly complex topography

What are the issues involved?

- Complex and often sparse geometries
- Large ranges in spatial and temporal scales
- Moving fronts and highly complex mixing zones

What do we hope to provide with such a tool?

- An enhanced ability to interpret and extend the results of field and laboratory studies
- A predictive tool for both engineering and science

## Variable Density Incompressible Navier-Stokes Equations

- Momentum balance

$$\vec{u}_t + (\vec{u} \cdot \nabla) \vec{u} = -\frac{\nabla p}{\rho} + \vec{g} + \nu \Delta \vec{u}$$

- Divergence free constraint

$$\nabla \cdot \vec{u} = 0$$

- Density conservation

$$\rho_t + \vec{u} \cdot \nabla \rho = 0$$

- Passive scalar transport

$$c_t + \vec{u} \cdot \nabla c = \nabla \cdot (k_c \nabla c) + H_c$$

Note that we do not employ Boussinesq or hydrostatic approximations.

## Solution Strategy: Temporal Discretization

We build on a classic second-order accurate projection method (Bell, Colella, Glaz, JCP 1989). We split the momentum equations into three pieces:

- Hyperbolic:  $\vec{u}_t + (\vec{u} \cdot \nabla)\vec{u} = H$

where we exactly enforce a divergence free state for the advective velocities, and compute the advective term explicitly

- Parabolic:  $\vec{u}_t = \nu \Delta \vec{u} + S$

which we solve implicitly for a predictor velocity

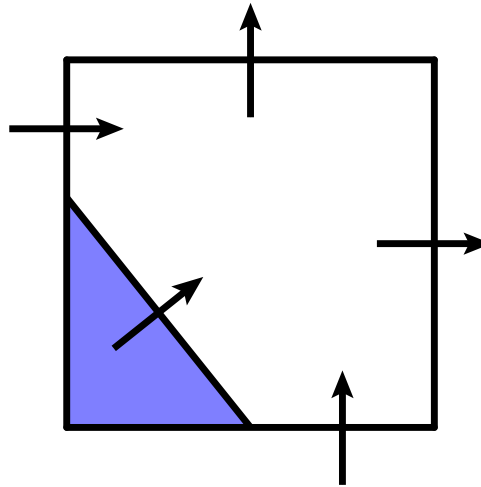
- Elliptic:  $\nabla \cdot \frac{1}{\rho} \nabla p = \nabla \cdot (-(\vec{u} \cdot \nabla)\vec{u} + \nu \Delta \vec{u})$

which we solve implicitly for pressure, and subsequently correct the predictor velocity

To update the scalar equations we do similar hyperbolic and parabolic decompositions.

### **Solution Strategy: Spatial Discretization Using Embedded Boundaries (EB)**

For the bulk of the flow,  $O(n^3)$  cells in 3D, we compute on a regular Cartesian grid. We use an embedded boundary description for the  $O(n^2)$  (in 3D) control-volumes that intersect the boundary.



Advantages of underlying rectangular grid:

- Grid generation is tractable, with a straightforward coupling to block-structured adaptive mesh refinement (AMR)
- Good discretization technology, e.g. well-understood consistency theory for finite differences, geometric multigrid for solvers.

## EB Finite-Volume Discretization Concepts

- Consider hyperbolic and elliptic conservation laws:

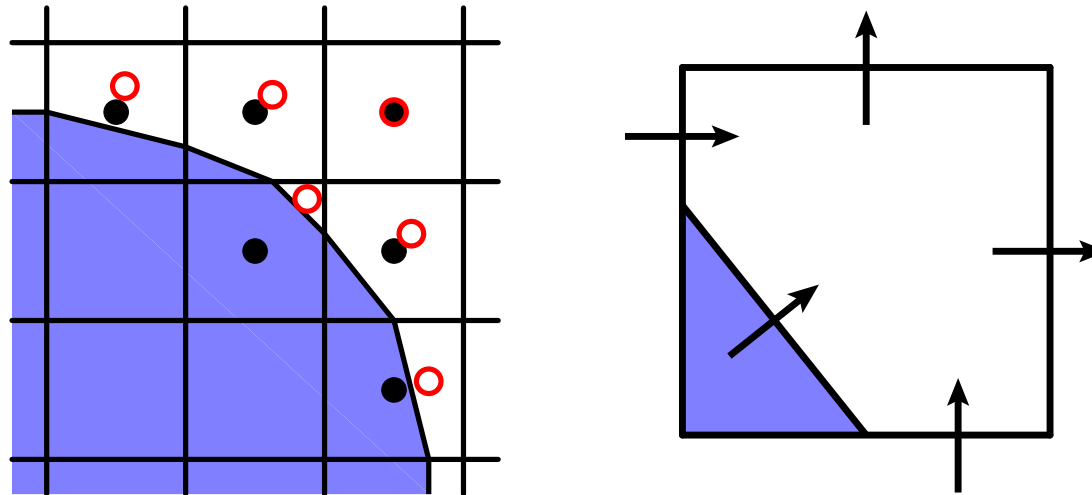
$$\frac{\partial U}{\partial t} + \nabla \cdot \vec{F}(U) = S \quad \text{and} \quad \nabla \cdot \nabla \phi = \nabla \cdot \vec{F} = H$$

- Primary dependent variables approximate values at centers of Cartesian cells
- Divergence theorem over each control volume leads to a “finite volume” approximation for  $\nabla \cdot \vec{F}$  (fluxes are at centroids):

$$\nabla \cdot \vec{F} \approx \frac{1}{\kappa \Delta x^D} \int \nabla \cdot \vec{F} dx = \frac{1}{\kappa \Delta x} \sum \alpha_s \vec{F}_s \cdot \vec{n}_s + \alpha_B \vec{F} \cdot \vec{n}_B \equiv D \cdot \vec{F}^c$$

Away from boundaries, our method reduces to a standard conservative finite difference method

- Given  $\kappa$ ,  $\alpha_s$ ,  $\alpha_B$ , and  $\vec{n}_B$  on a fine grid, we can compute these on coarser grids without reference to the original geometry.



## EB Finite-Volume Discretization Concepts, cont.

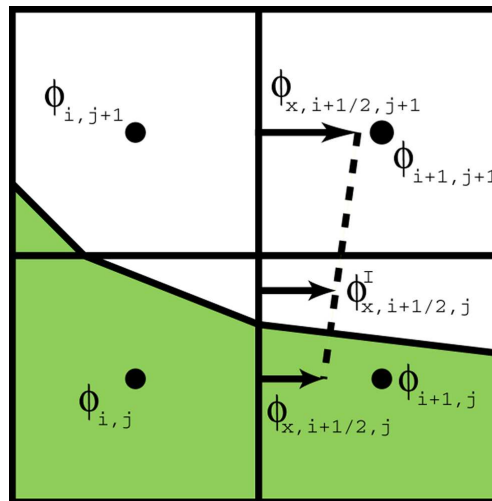
For the case of an elliptic conservation law, we need to compute  $\nabla\phi$  at face centroids. Here is how we do this in 2D:

- First we compute face-centered gradients using centered differences

$$\phi_{x,i+1/2,j} = \frac{\phi_{i+1,j} - \phi_{i,j}}{\Delta x}$$

$$\phi_{x,i+1/2,j+1} = \frac{\phi_{i+1,j+1} - \phi_{i,j+1}}{\Delta x}$$

- Then we linearly interpolate the gradient (or flux) to the face centroid, yielding  $\phi_{x,i+1/2,j}^I$

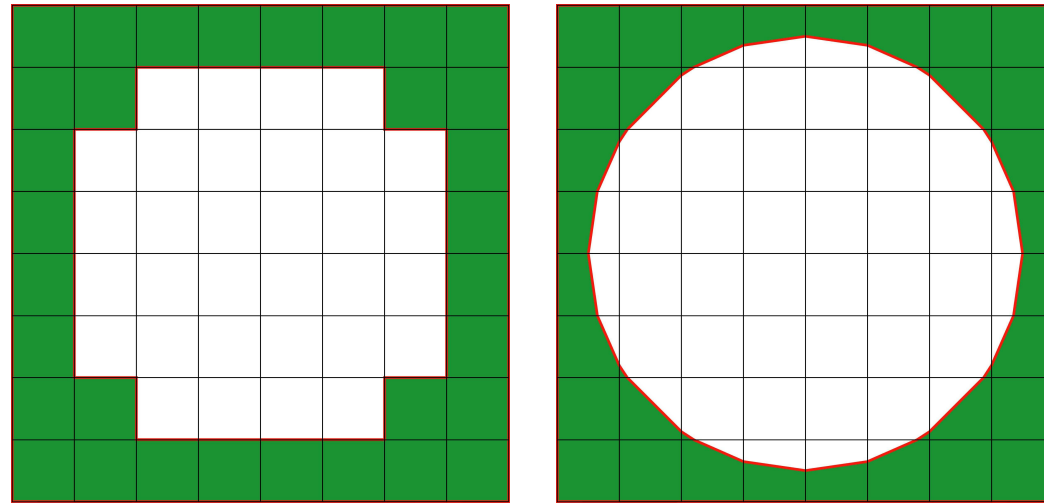


- In 3D we use a bilinear interpolation based on four face-centered gradients (fluxes)



## Why are Embedded Boundaries Important?

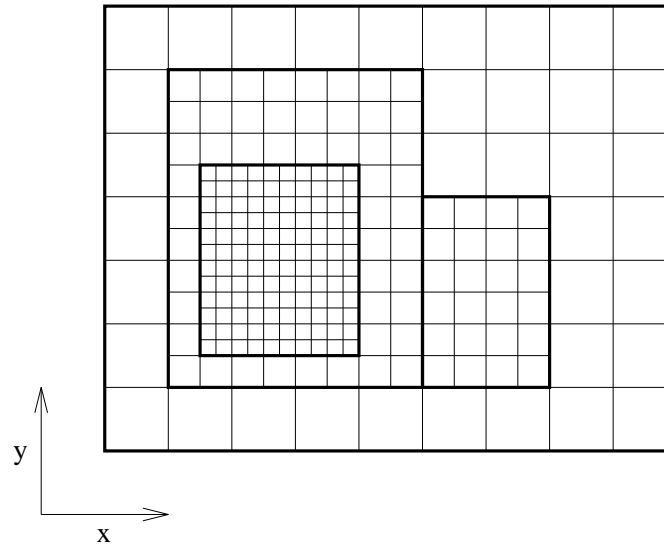
To accurately discretize our conservation laws all we need are the following quantities: volume fractions, area fractions, centroids, boundary areas, and boundary normals.



Above are two approximations to a circle: stair-step on left, and EB on right.

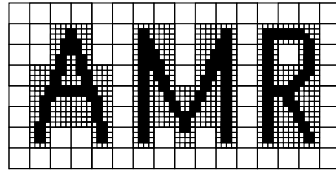
	Stair-Step	Embedded Boundary
Area Error	$O(h)$	$O(h^2)$
Perimeter Error	$O(1)$	$O(h^2)$
Boundary Normal Errors	$O(1)$	$O(h^2)$

## Block-Structured Adaptive Mesh Refinement



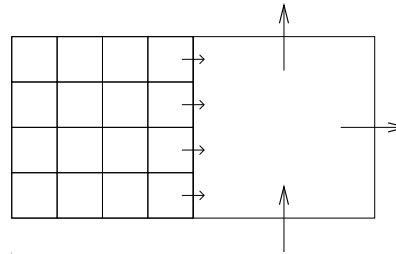
In adaptive methods, one adjusts the computational effort locally to maintain a uniform level of accuracy throughout the problem domain.

- Refined regions are organized into rectangular patches. Refinement is possible in both space and time.
- AMR allows the simulation of a range of spatial and temporal scales. Capturing these ranges is critical to accurately modeling multi-scale transport complexities such as boundaries, fronts, and mixing zones that exist in natural environments.
- We maintain accuracy and strict conservation with embedded boundaries and AMR

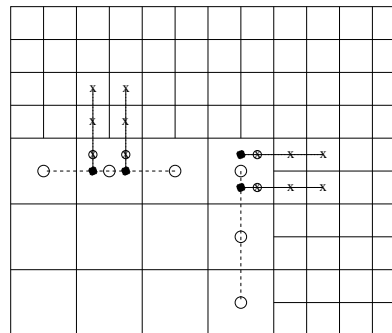


## Finite-Volume Discretization Concepts

- Two requirements are necessary to maintain conservation and second-order accuracy with AMR:
  - 1) Match fluxes conservatively at coarse fine interfaces (this leads to a refluxing step for the coarse levels)

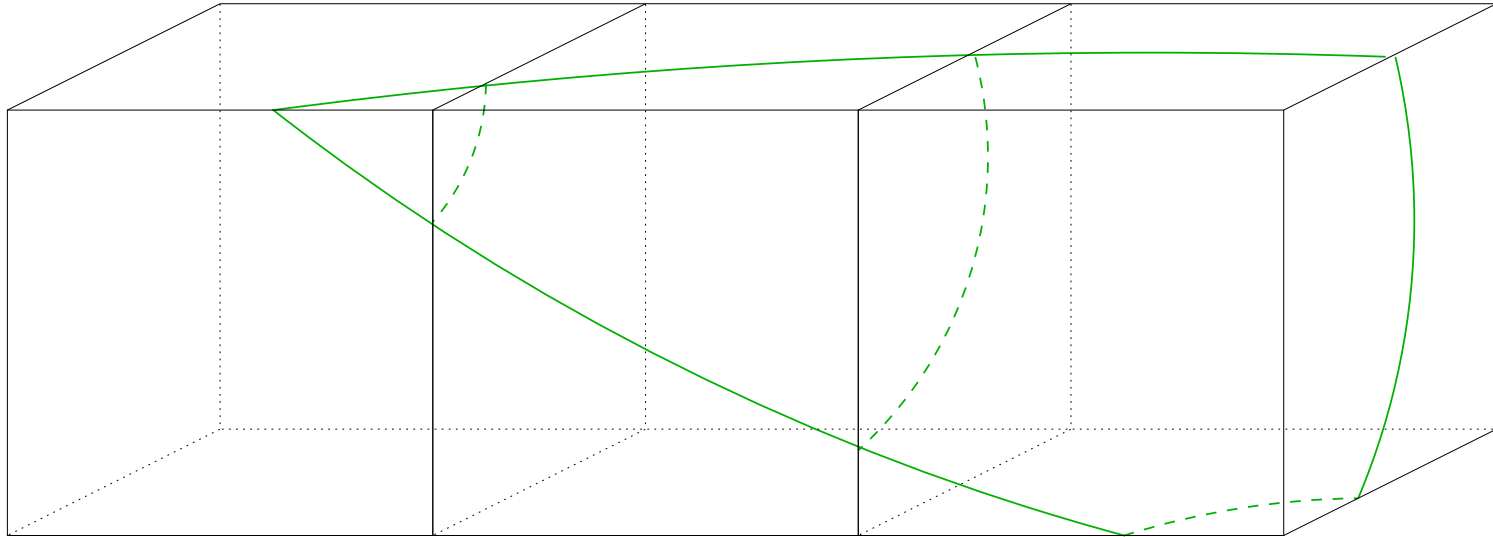


- 2) Use quadratic interpolation to provide ghost cell values for points in the stencil extending outside of the grids at that level



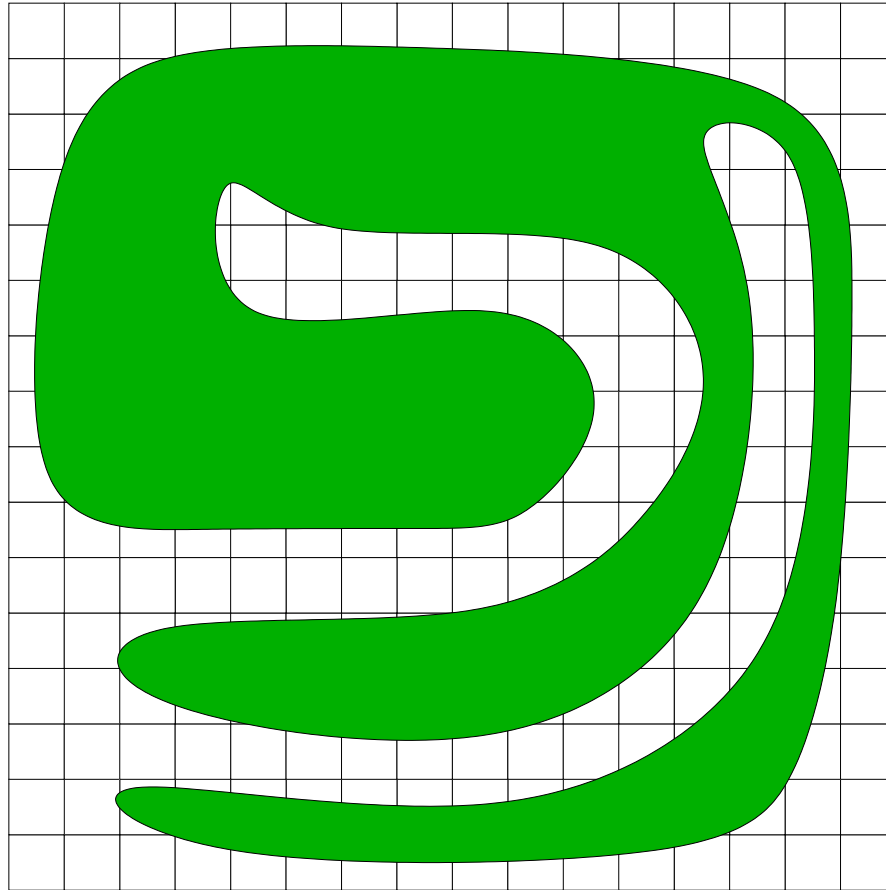
## Grid Generation for Embedded Boundaries

- Three example irregular cells are shown below. Green curves indicate the intersection of the exact boundary with a Cartesian cell. We approximate face intersections using quadratic interpolants.



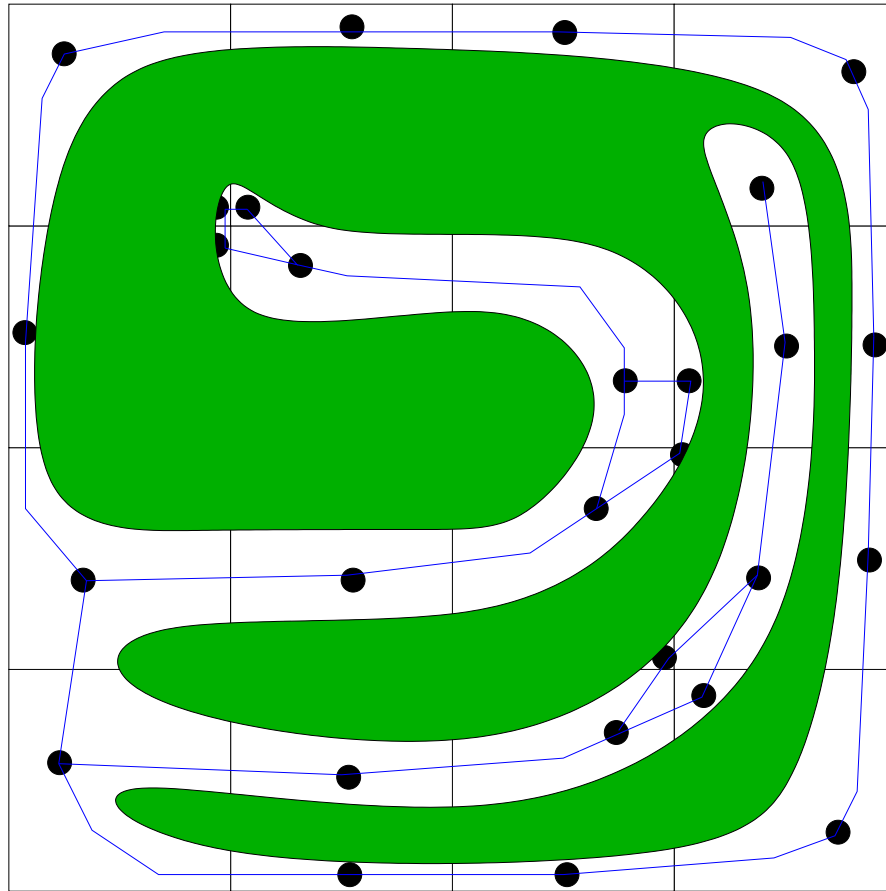
- EB's are “water tight” by construction, i.e. if two control-volumes share a face, they both have the same area fraction for that face.
- Unlike typical discretization methods, the EB control volumes naturally fit within easily parallelized disjoint block data structures.
- Permits dynamic coarsening and refinement of highly complex geometry as a simulation progresses.

## EB Coarsening



White regions represent individual control volumes. Green is the exact geometry.

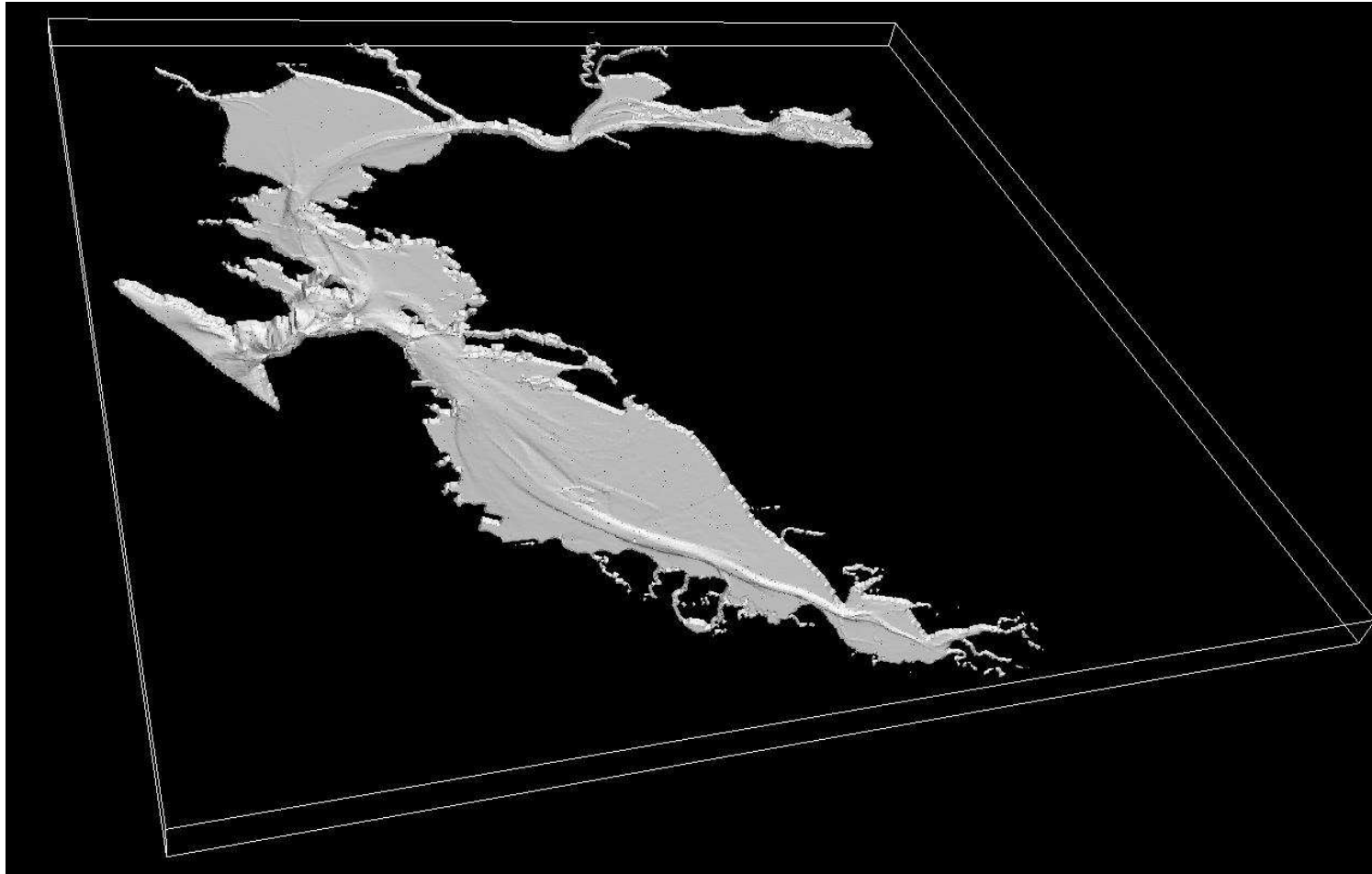
## EB Coarsening



Note: We have some cells with more than one control volume per cell.

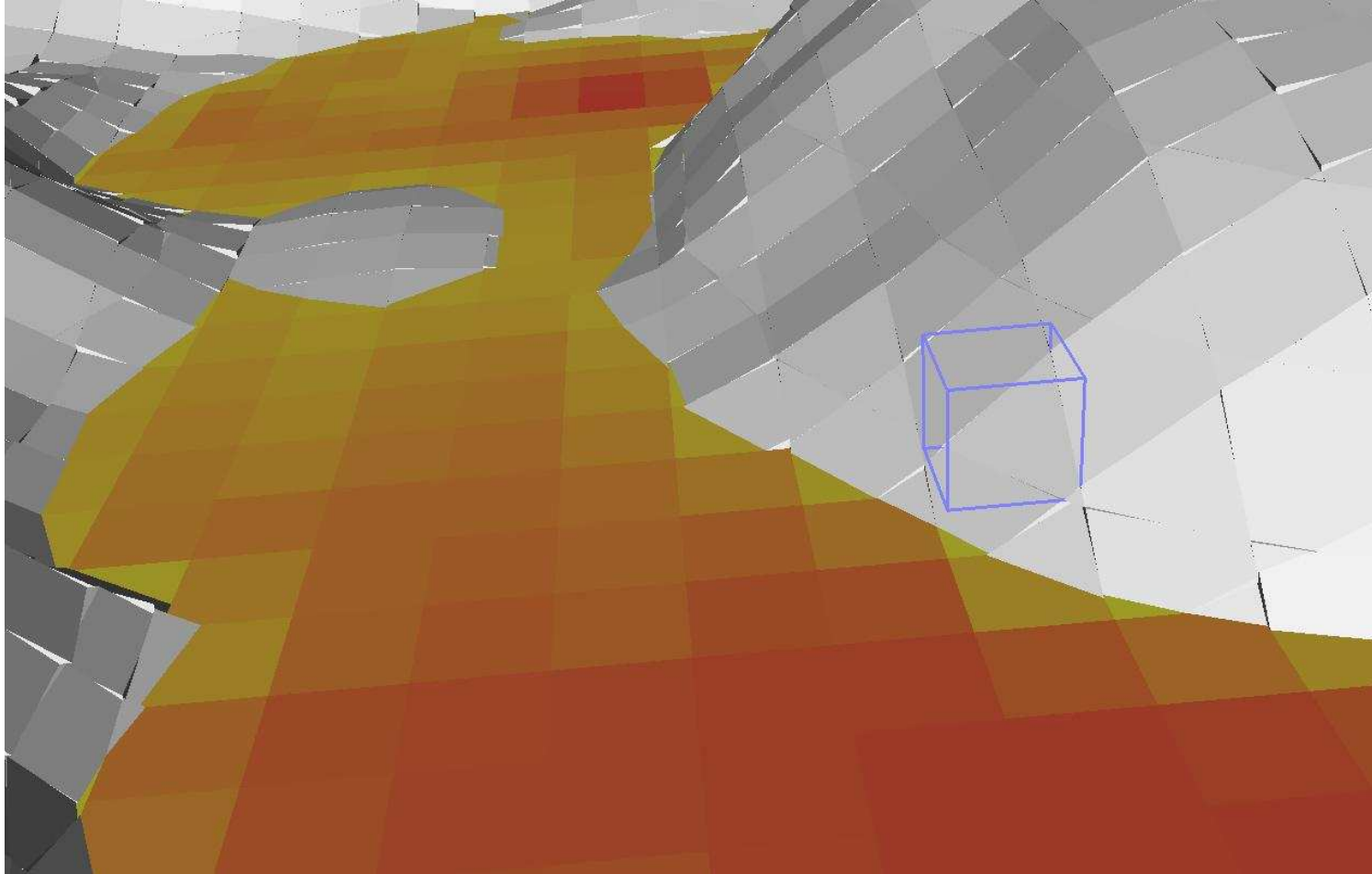
## EB Grid Generation Examples

- Embedded boundary description of San Francisco Bay (using USGS DEM data)



## EB Grid Generation Examples

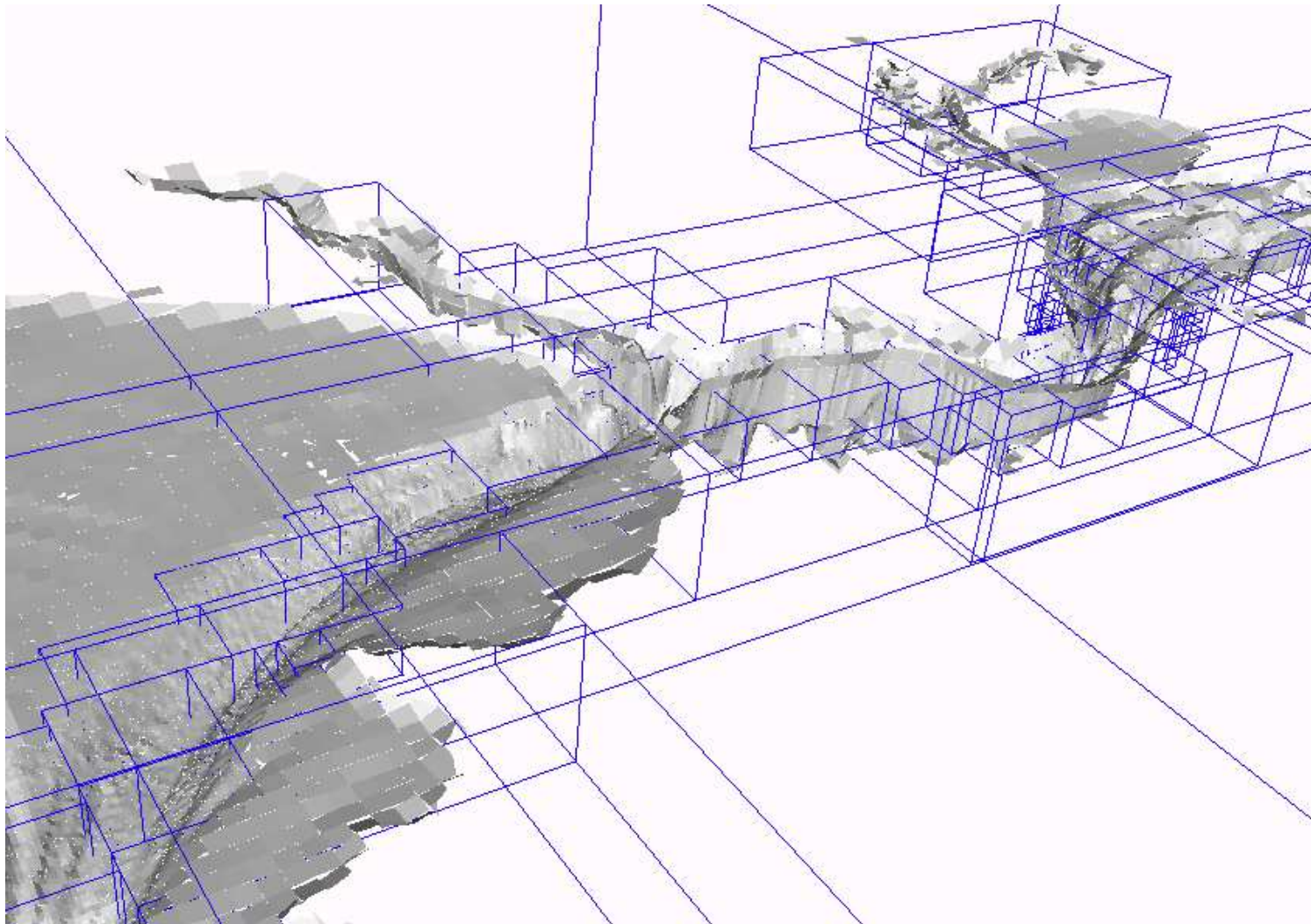
- Embedded boundary description of San Francisco Bay (using USGS DEM data)





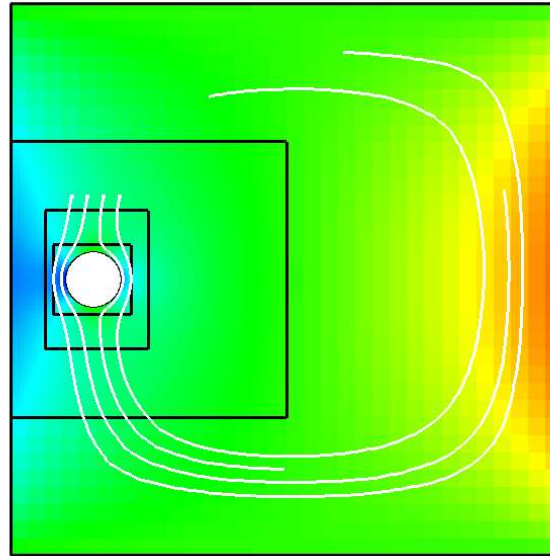
## EB AMR Grid Generation Examples

- San Francisco Bay with AMR (using USGS DEM data)



## Results: 2D Convergence Study

- Flow is inside a 1 m square tank, with a cylinder of diameter 0.1m at (0.15,0.5)
- The initial velocity field is the divergence free part of a rigid rotation (flow is counter-clockwise)



Above is a plot of the initial conditions. The colors indicate vertical velocity (red is up, blue is down)

- The  $Re = 100$  for this problem.
- We initialize our density field as  $\rho = 1000 + 30y$  (linear, with heavy on top)

## Results: 2D Convergence Study

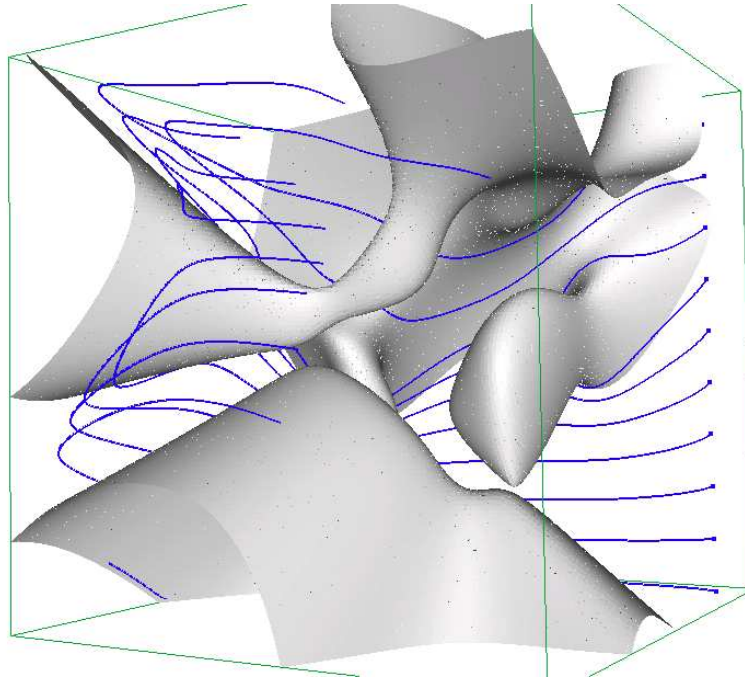
- For the convergence study, we solve this problem on 3 different grid hierarchies. We discretize the coarsest level of each hierarchy with 8x8, 16x16, and 32x32 cells.
- For each run we use 4 levels of AMR with a refinement ratio of 4 between levels
- In the table we present norms of the solution error, and our convergence rate.

Base Grids	8-16	Rate	16-32
$L_1$ Norm of V Velocity Error	5.73e-3	<b>2.02</b>	1.42e-3
$L_2$ Norm of V Velocity Error	7.68e-3	<b>2.01</b>	1.91e-3
$L_1$ Norm of Scalar Error	1.24e-2	<b>2.04</b>	3.03e-3
$L_2$ Norm of Scalar Error	1.64e-2	<b>1.98</b>	4.16e-3

## Results: 3D Convergence Study

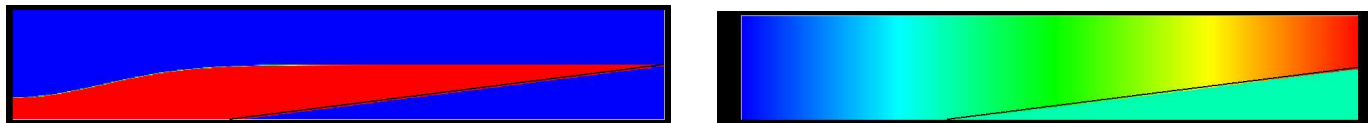
- Below is a 3D single level convergence study for a constant density,  $Re = 100$ , rotational flow past a complex geometry:

Base Grids	16-32	Rate	32-64
$L_1$ Norm of U Velocity Error	1.69e-2	<b>2.32</b>	3.39e-3
$L_2$ Norm of U Velocity Error	5.28e-2	<b>1.76</b>	1.55e-2
$L_1$ Norm of W Velocity Error	1.48e-2	<b>2.29</b>	3.03e-3
$L_2$ Norm of W Velocity Error	4.69e-2	<b>1.83</b>	1.32e-2



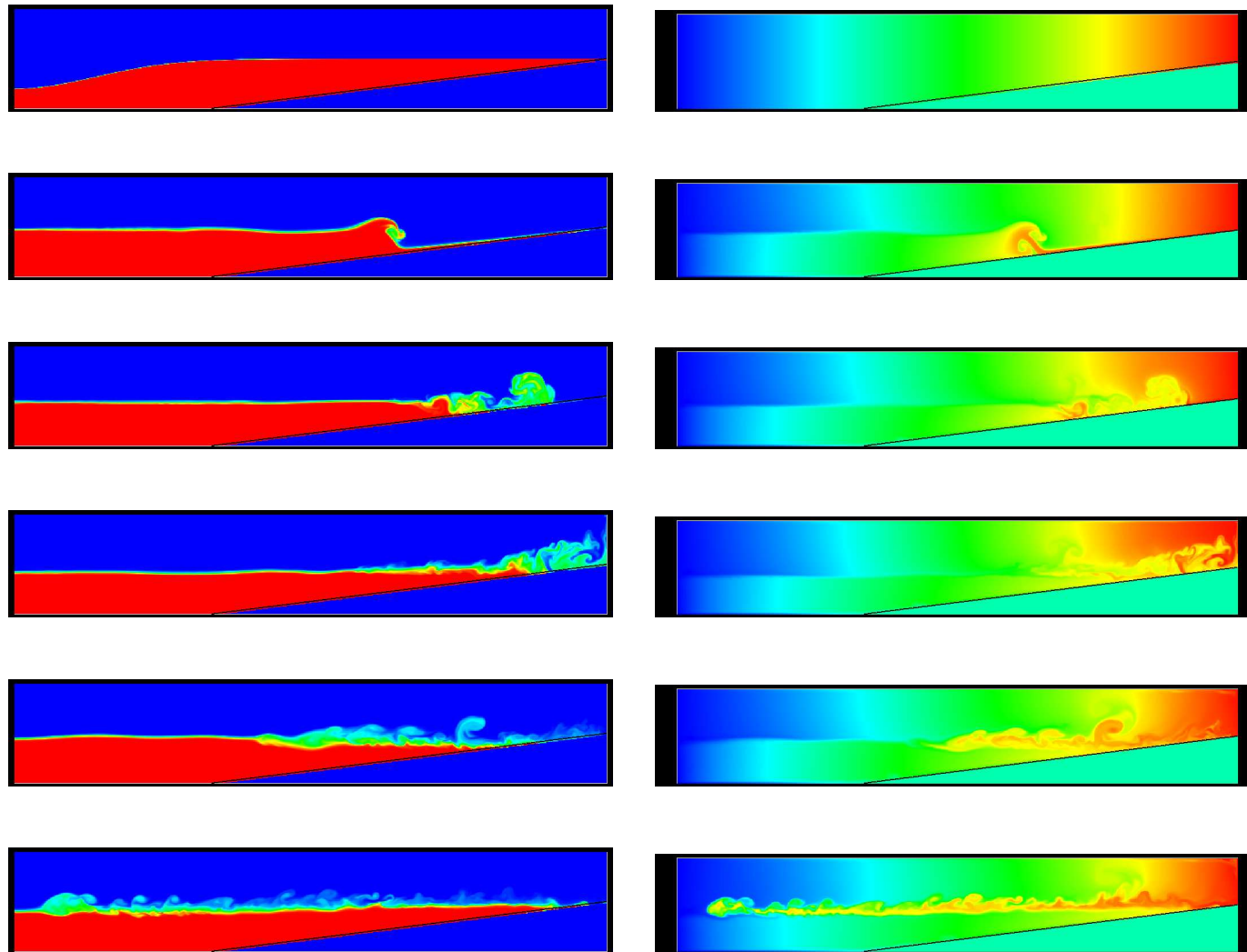
## Results: Breaking Internal Waves on a Slope

- Flow is inside a 0.5m tall, by 3m wide tank, with an 8:1 slope starting 1m from the left side
- Below left is the initial density distribution (blue is light fluid, red is heavy fluid), below right is the initial conditions for a passive scalar



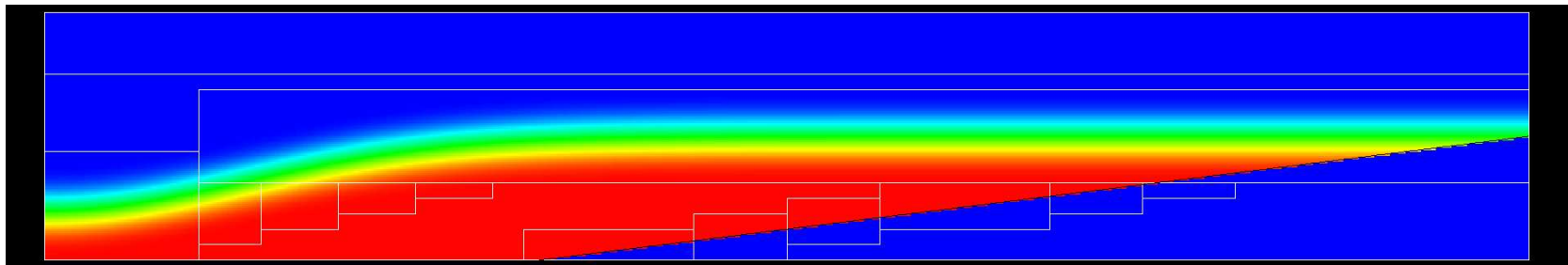
- Density ratio of light fluid to heavy fluid is  $1000/1030$ , and our pycnocline is a step-function. The pycnocline is perturbed on the left side of the tank.
- Thanks to Prof. Fringer for this test problem

## Breaking Internal Wave on a Slope (Density left, Scalar right)



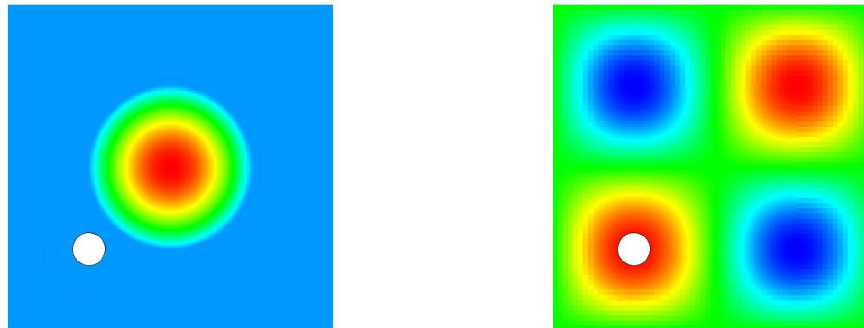
### Results: Breaking Internal Waves on a Slope

- Since most pycnoclines are not step-functions, what happens if our initial pycnocline is smoothed over 20 percent of the depth?



### Results: Simulation of flow past a cylinder with AMR

- Flow is inside a 1m square tank, with a cylinder of diameter 0.1m placed at (0.25,0.25)
- We initialize the flow with a vortex patch in the center of the domain, see below left. The  $Re = 100$  for this flow. We initialize a passive scalar, see below right.

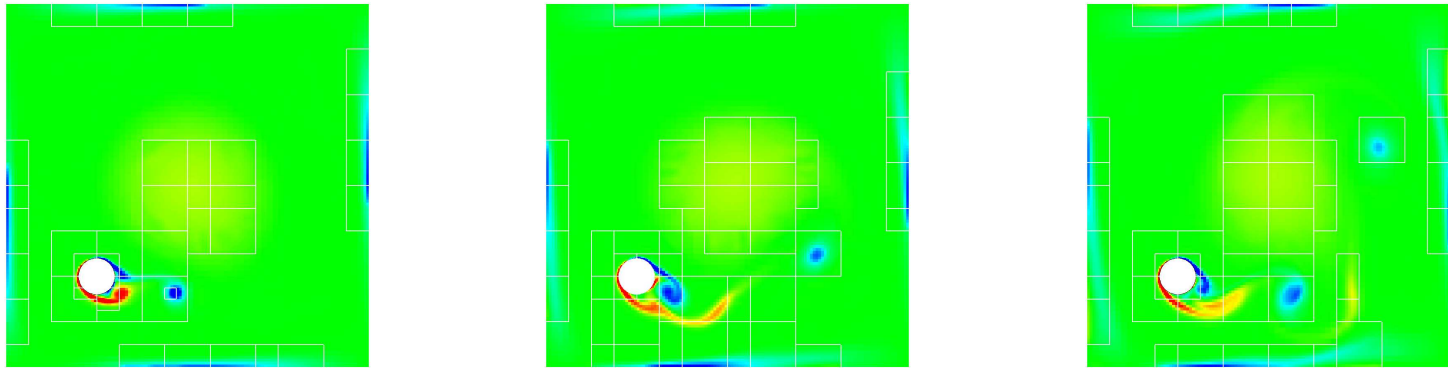


- We discretize the domain with a base grid of 64x64 cells, and refine 2 additional levels to track vorticity

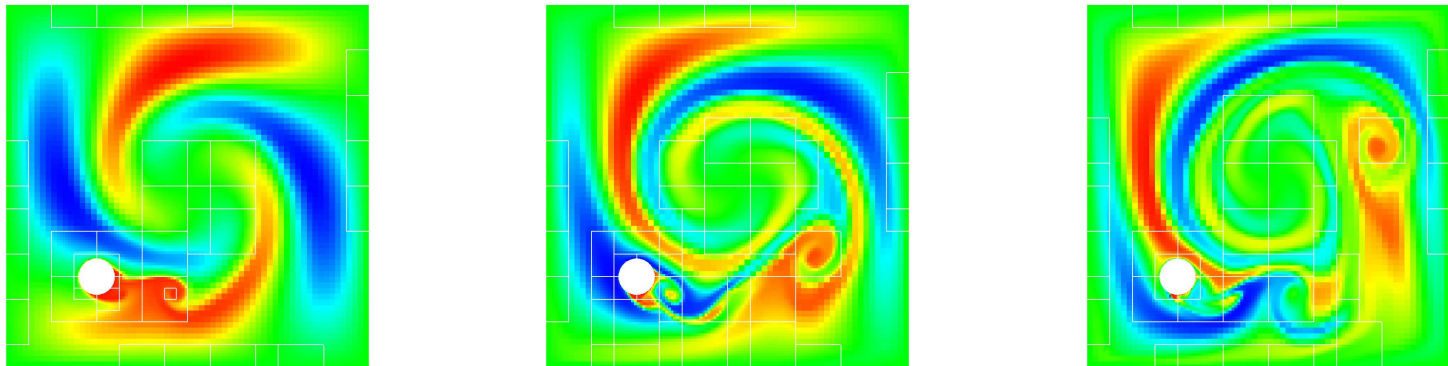


## Simulation of flow past a cylinder with AMR

- Vorticity (which we track with AMR):

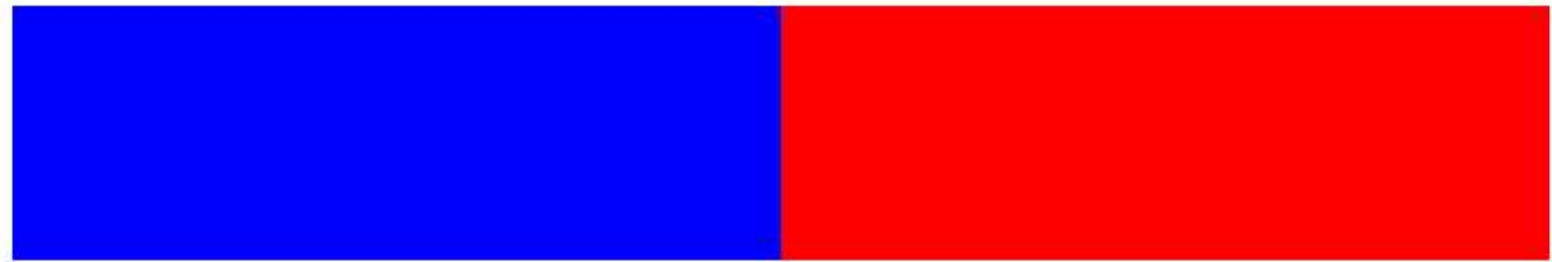


- Passive Scalar:



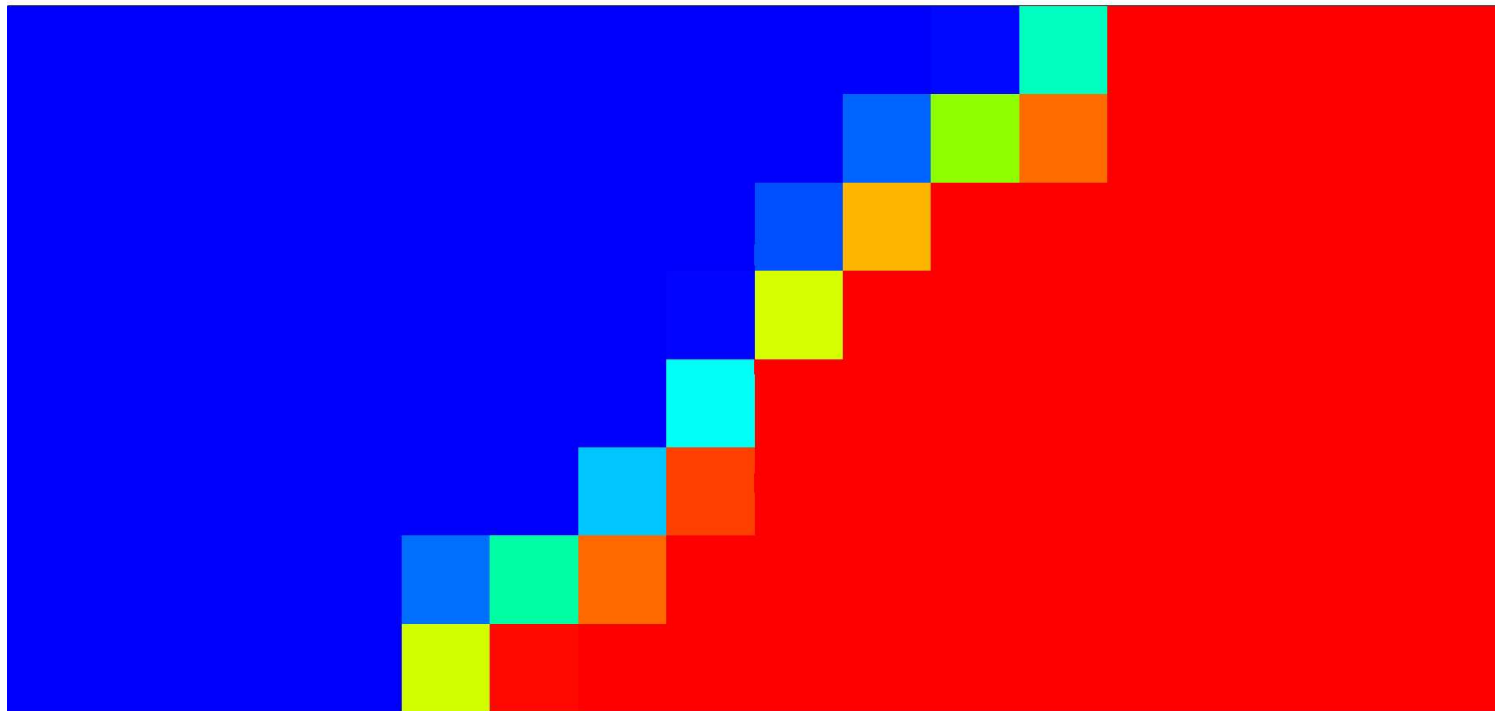
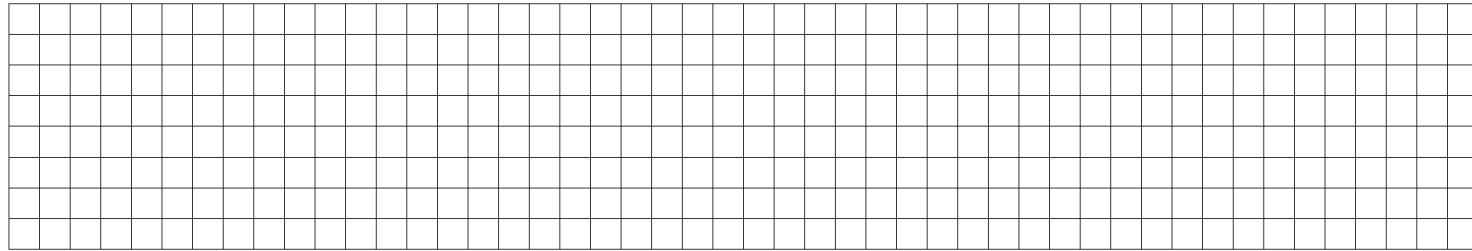
### Results: Lock-Exchange with AMR

- Flow is inside a 0.5m tall, by 3m wide tank.
- On the left side of the tank we start with light water, on the right is heavy water. The density ratio of light fluid to heavy fluid is 1000/1030.

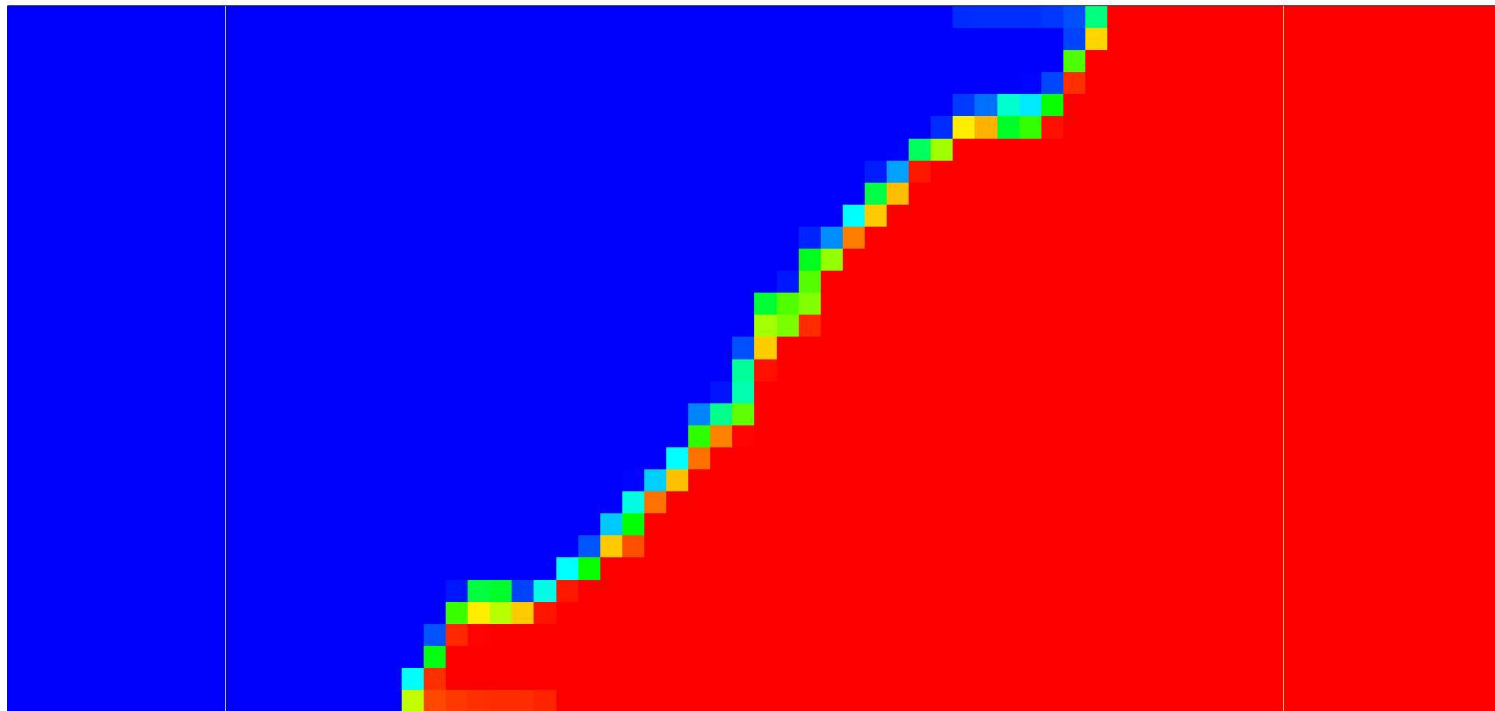
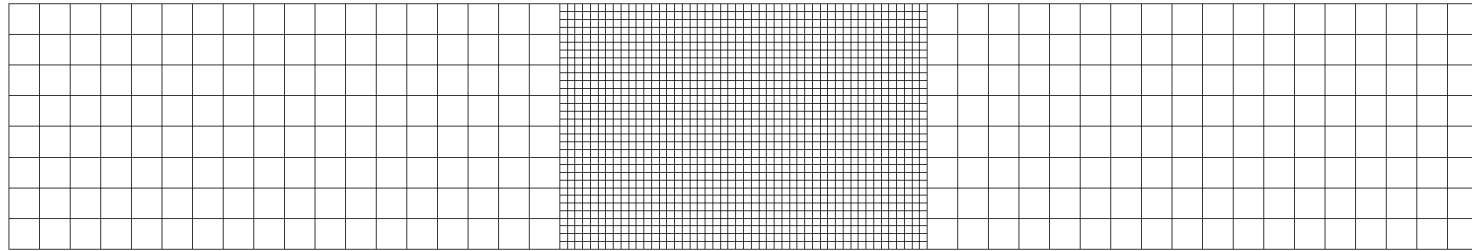


- On the following lock-exchange slides, the lower figure is a zoom in on the center region of the tank.

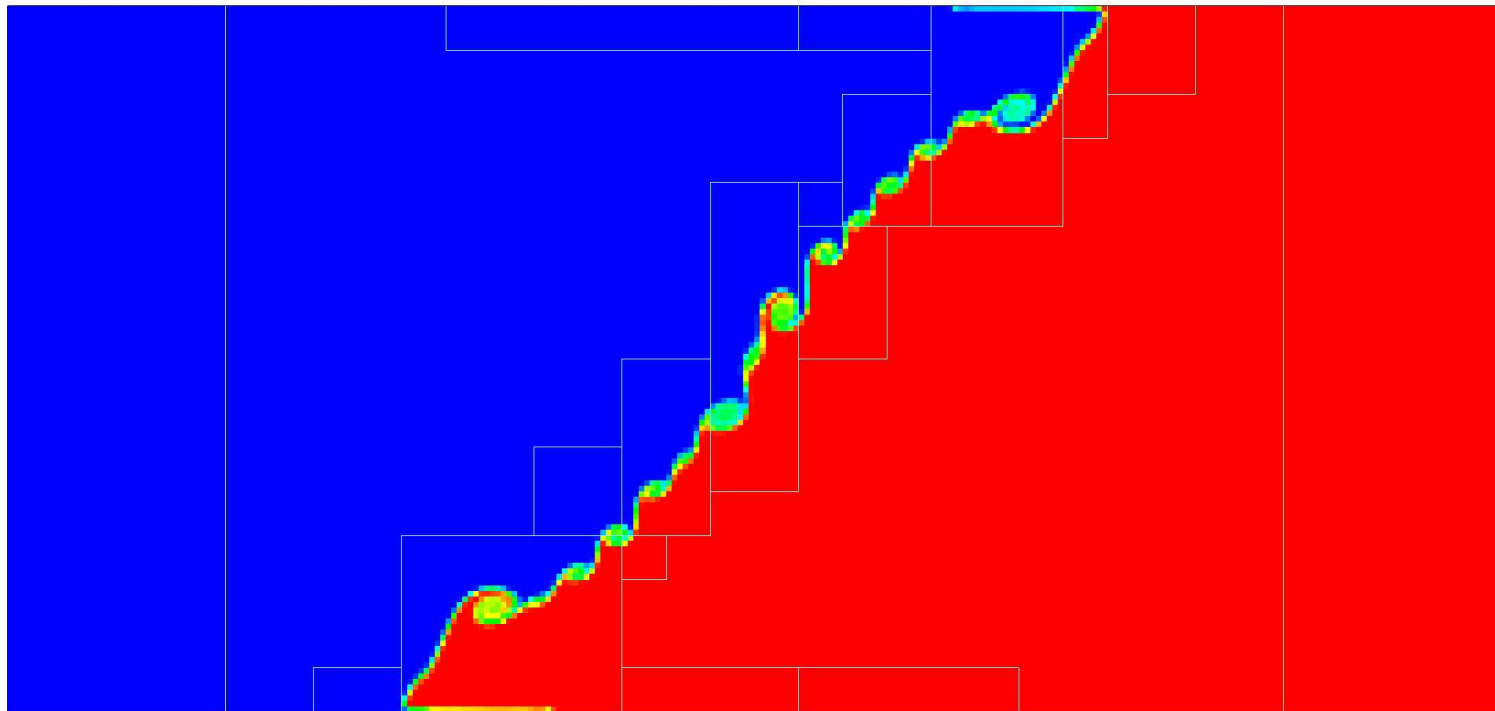
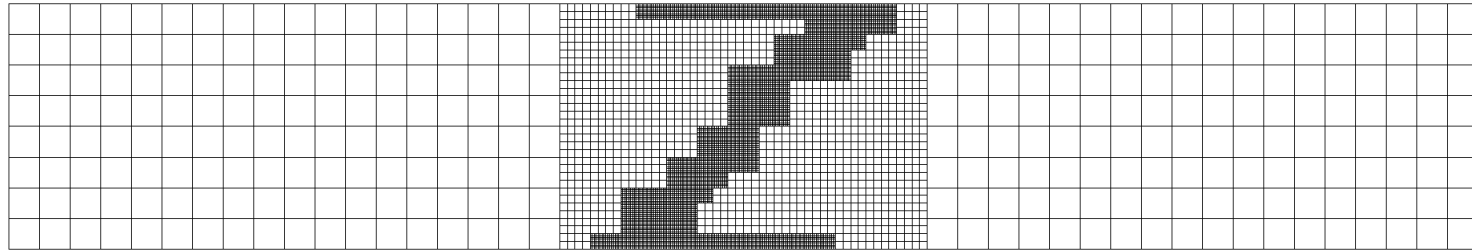
## Lock-Exchange: Why is AMR important?



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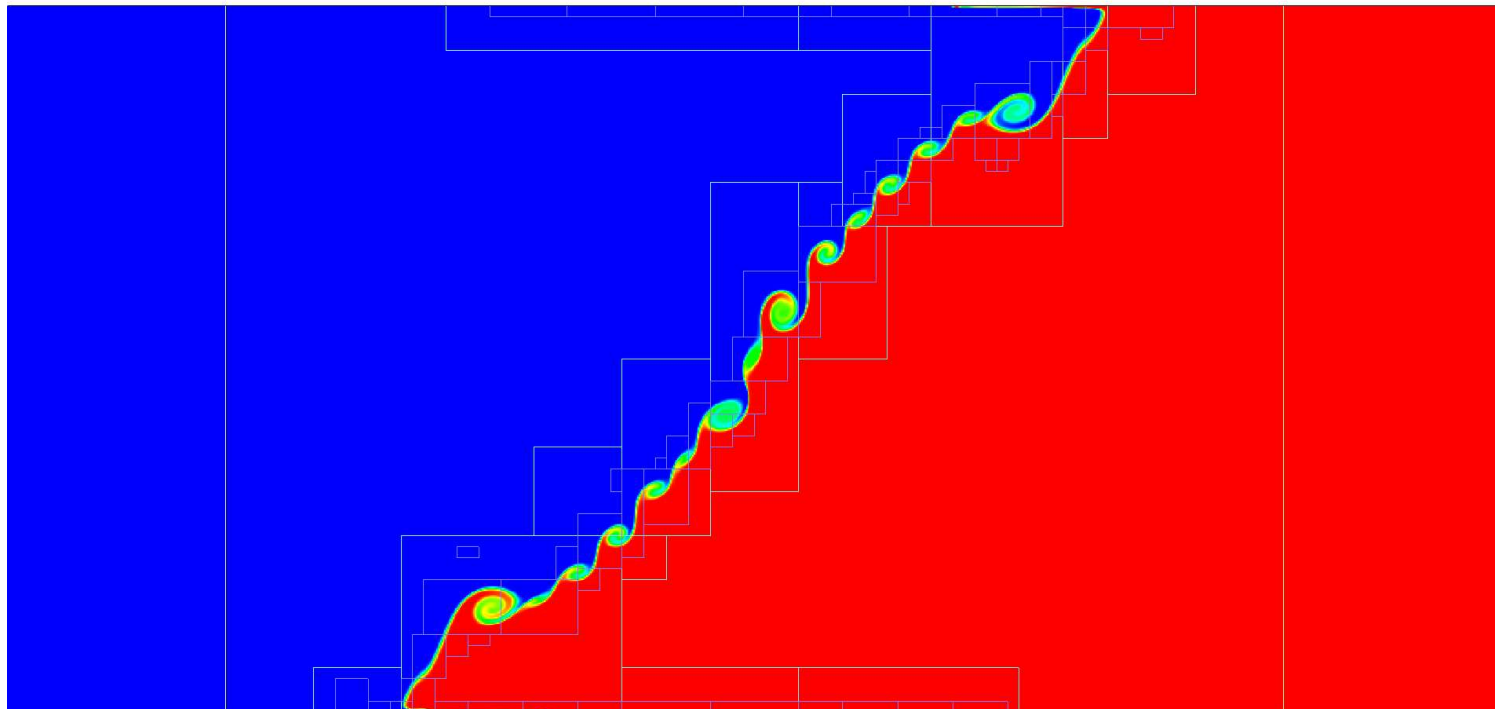
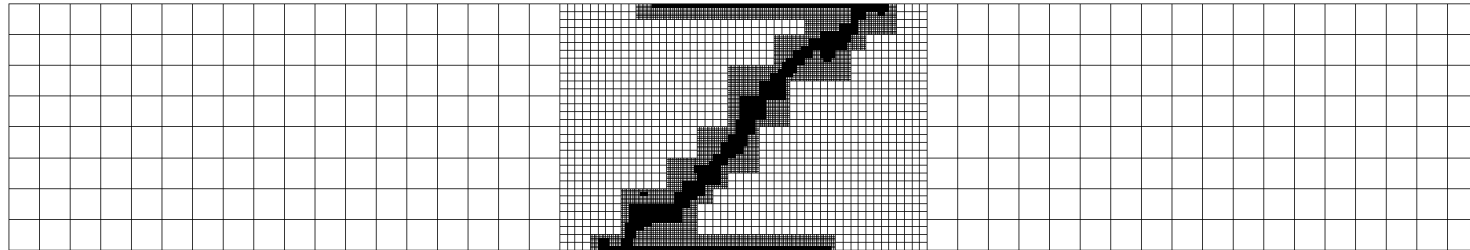


## Lock-Exchange: Why is AMR important?



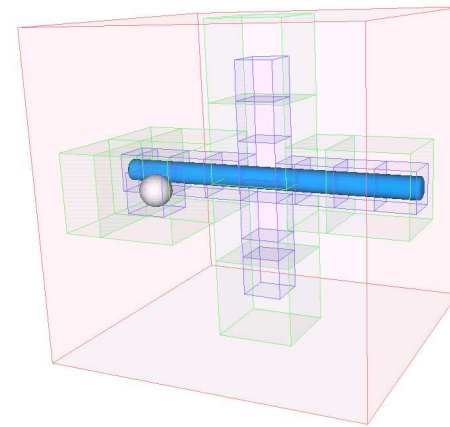
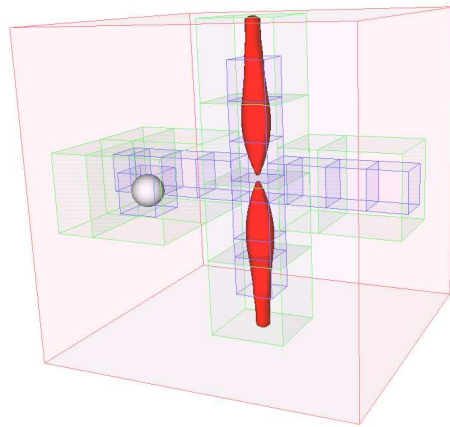
**Lock-Exchange: Why is AMR important?**

**Answer: We can add computational effort only where we need it!**



## Results: Simulation of flow past a sphere with AMR

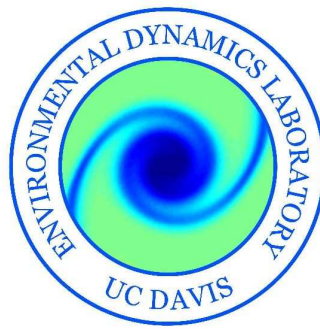
- Flow is inside a 1m cubed tank, with a sphere of diameter 0.1m placed at (0.25,0.25,0.5)
- We initialize the flow with a vortex patch in the center of the domain, see below left for z-vorticity iso-surfaces. The  $Re = 100$  for this flow. We initialize a passive scalar, see below right for an iso-surface.



## Conclusions and Future Work

- We now have a second-order accurate incompressible Navier-Stokes code that has been validated in 2D and 3D.
- Our AMR version is showing reasonable results and is under review to ensure second-order accuracy.
- Future Work:
  - Free Surface Tracking
  - LES Turbulence Closure
  - Field Scale Applications
  - Fourth-Order Accuracy (see my upcoming JCP paper with P. Colella).





## Acknowledgments

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- This research is funded by the Computational Science Graduate Fellowship program of the Department of Energy.
- Check out my dusty web site:  
<http://edl.engr.ucdavis.edu/barad.html>